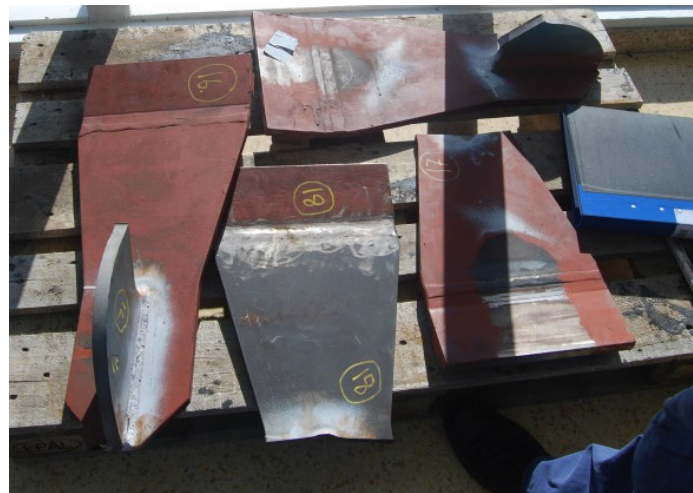


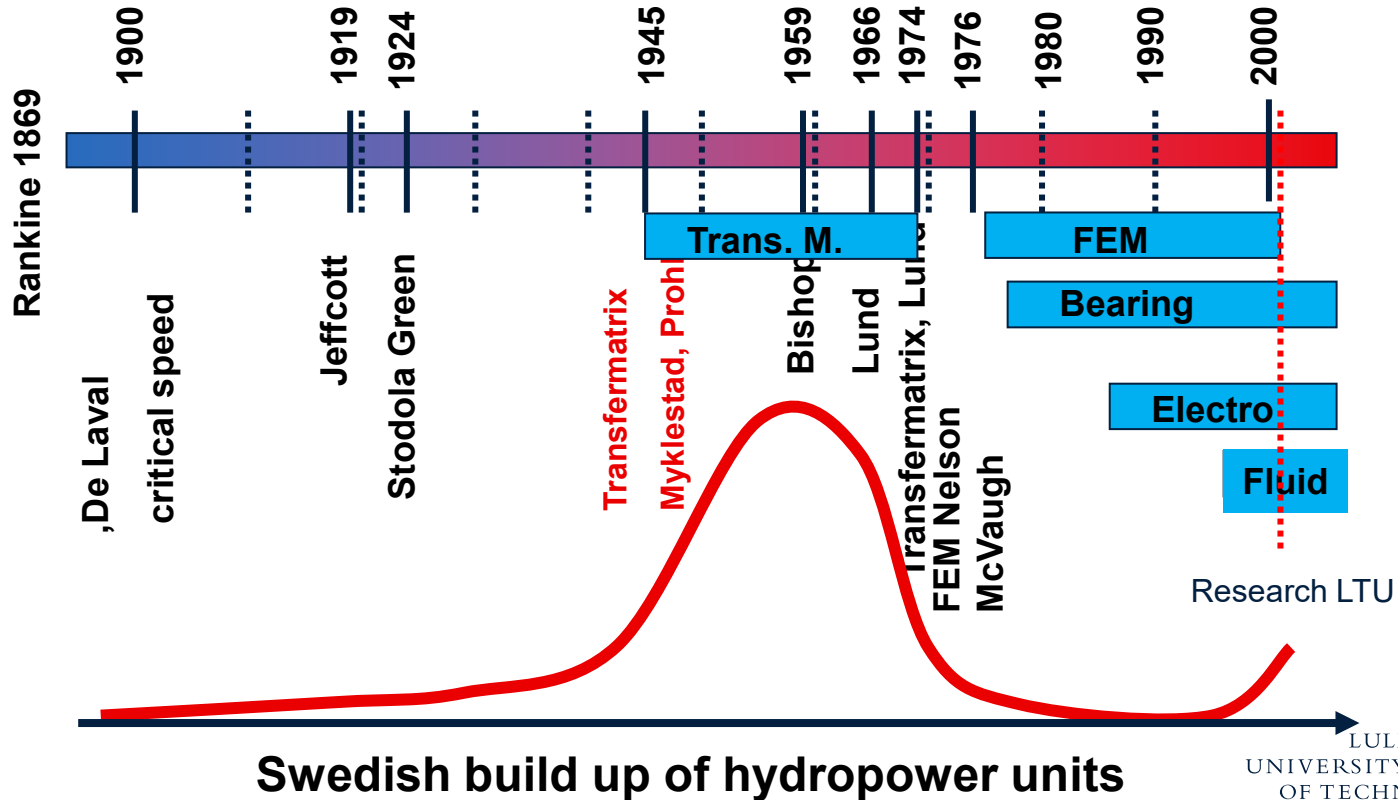
ROTOR DYNAMICS IN HYDROPOWER APPLICATIONS

Developments and Future in SVC

Prof. Jan-Olov Aidanpää, LTU

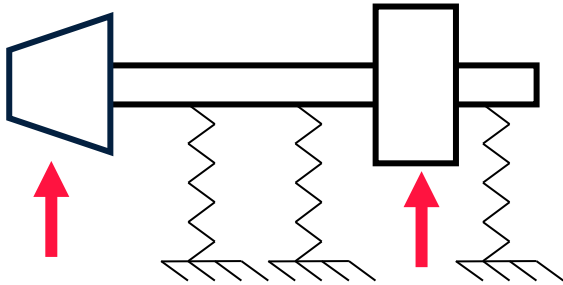


Rotordynamic History and Hydropower



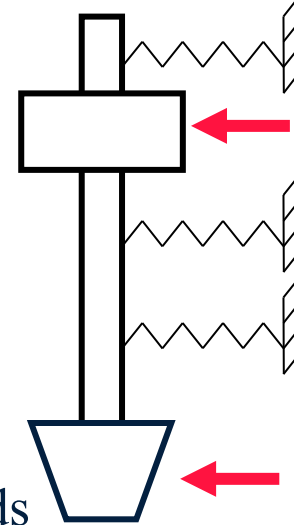
Why are Hydropower Rotors Different

Ordinary rotating machines



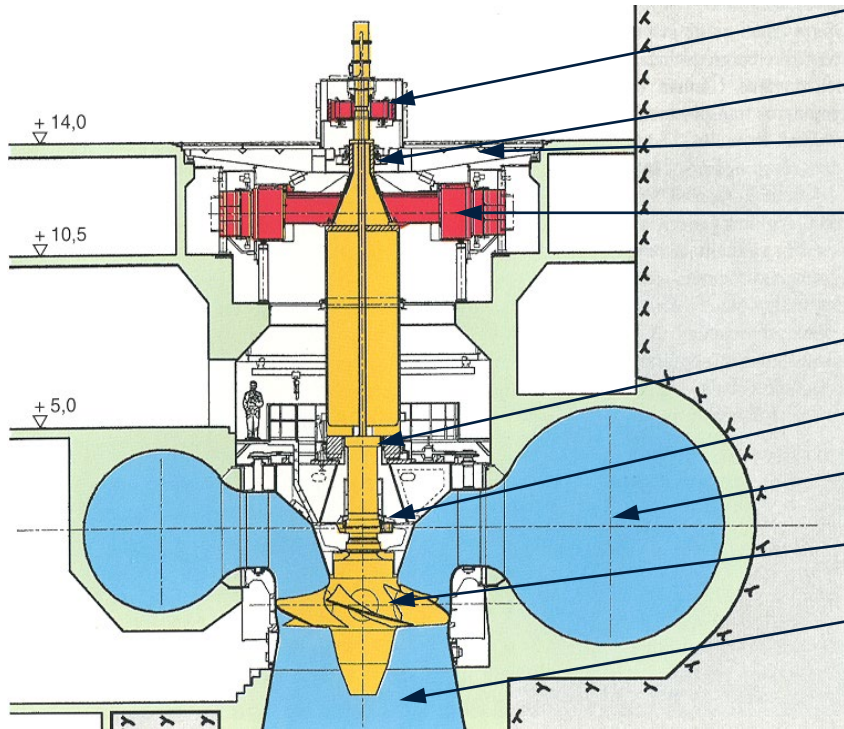
Known loads
known bearing prop
large air gap
small shape deviations

Hydro-power



Unknown loads
no relevant bearing models
small air gaps
large shape deviations

Hydropower Rotors are Unique



Exciter

Generator journal

Upper generator

Generator rotor

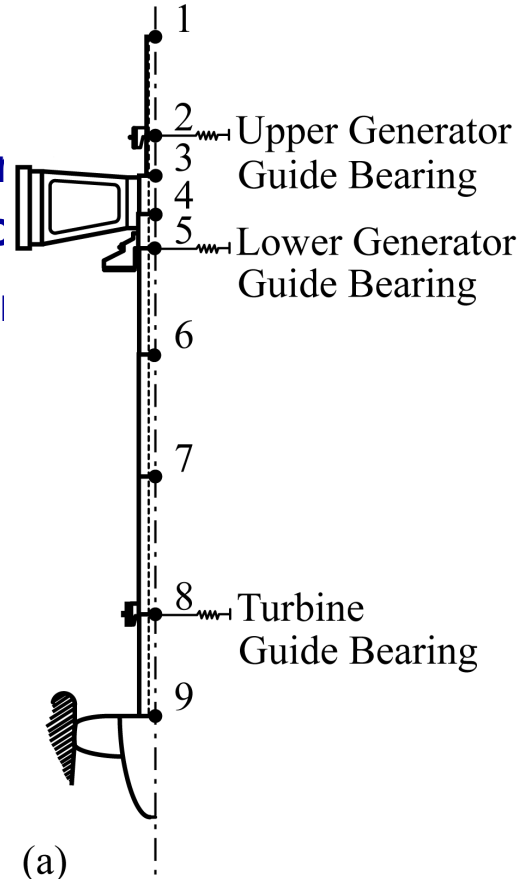
Trust bearing

Turbine journal

Spiral casing

Turbine

Draft tube



$$\frac{1}{K_{tot}} = \frac{1}{K_{film}} + \frac{1}{K_{mek}} \quad (1)$$

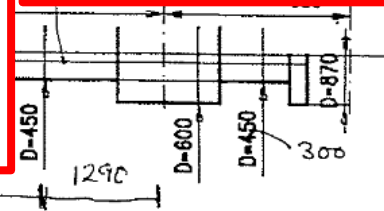
| Driftvarvtal | | | |
|--------------|---------------------|---------------------|-----------------------|
| | Upper guide bearing | Lower guide bearing | Turbine guide bearing |
| K_{film} | 2.6×10^6 | 1.16×10^6 | 0.90×10^6 |
| K_{mek} | 1.17×10^6 | 7.5×10^6 | 3.0×10^6 |
| K_{tot} | 0.8×10^6 | 1.0×10^6 | 0.69×10^6 |

Tabell 2.1.1. Minsta beräknade lagerstyvheter vid driftvarvtal [N/mm]. Oljefilmens styvheter för övre och nedre styrlager är tagna från [1]. Turbinlagerdata har erhållits från turbinleverantören.

| Rusvarvtal | | | |
|------------|---------------------|---------------------|-----------------------|
| | Upper guide bearing | Lower guide bearing | Turbine guide bearing |
| K_{film} | 2.7×10^6 | 1.6×10^6 | ∞ |
| K_{mek} | 1.17×10^6 | 7.5×10^6 | 3.0×10^6 |
| K_{tot} | 0.82×10^6 | 1.32×10^6 | 3.0×10^6 |

Tabell 2.1.2. Minsta beräknade lagerstyvheter vid rusvarvtal [N/mm]. Oljefilmens styvheter för övre och nedre styrlager är tagna från [1]. Turbinlagerdata har erhållits från turbinleverantören.

No Damping
 Unsure Stiffnesses
 No Reliable Results



F. Exciter rotor

Total weight = 1000 kg

CG : 690 kg

CG : 600 kg

D. Turbine guide bearing

Radial stiffness (proposed values by ABB)

Normal speed = 800 r/min

Mechanical ~~1.7E6~~ $3.0E6$ N/mm

Oil film $0.6+E6$ N/mm

Total ~~9.44E6~~ $0.9E6$ N/mm

Runaway speed = 1450 r/min

Mechanical $1.7+E6$ $3.0E6$ N/mm

Oil film ~~0.8+E6~~ N/mm

Total ~~0.544+E6~~ $3.0E6$ N/mm

$k_{mek} = 2.8 \cdot 10^9$ N/m = $k_{tot,ms}$

E. Turbine runner

Total weight 1850 kg

Weight incl. water ??? kg 2600

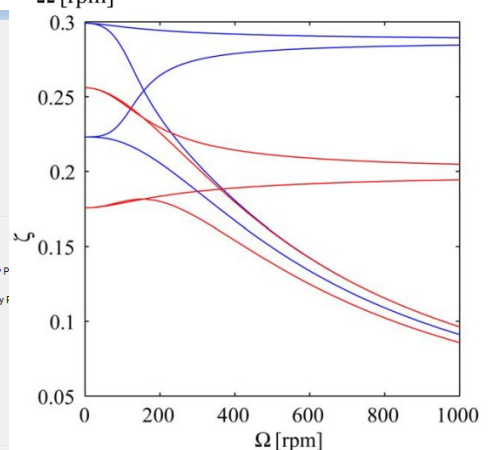
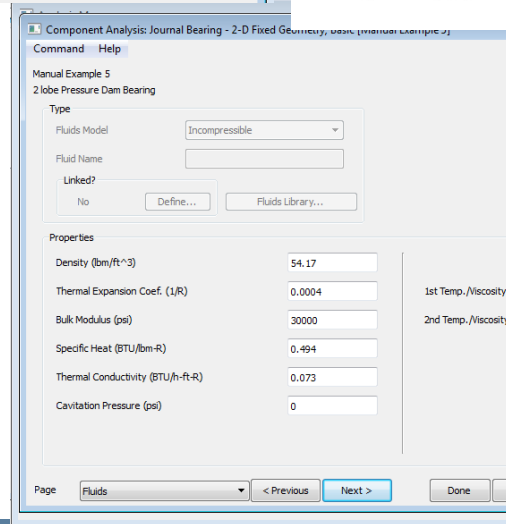
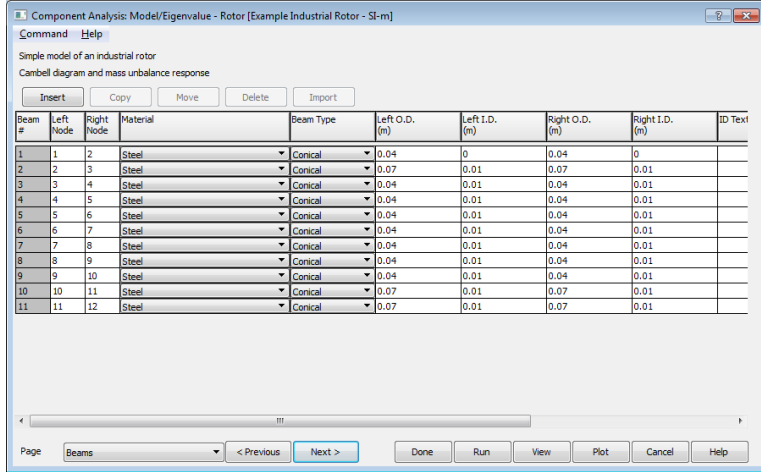
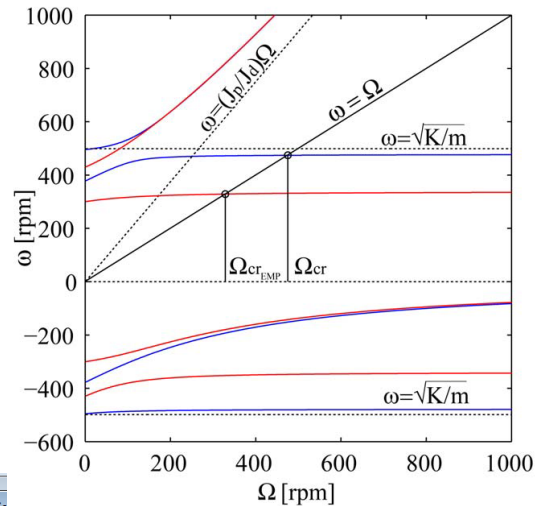
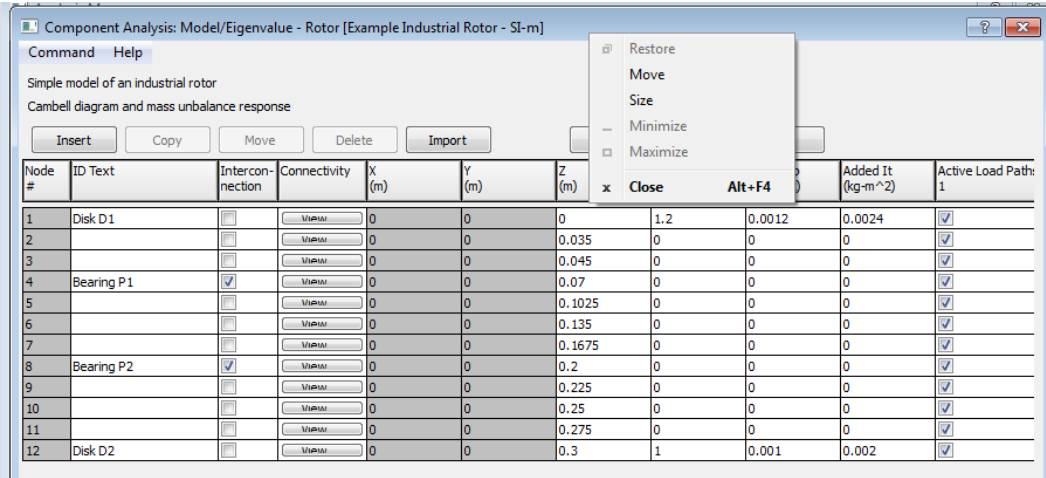
Moment of inertia (incl. water?)

Polar 200 kgm² 310

Dicmetrical ??? kgm² 230

Max. radial force on turbine runner-





Rotordynamics within SVC in Close Collaboration with VRD

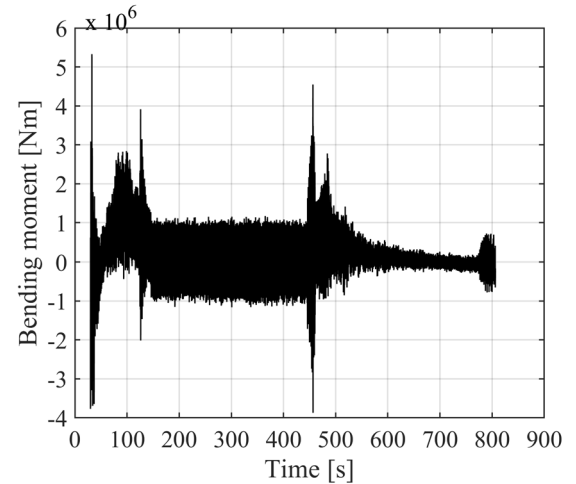
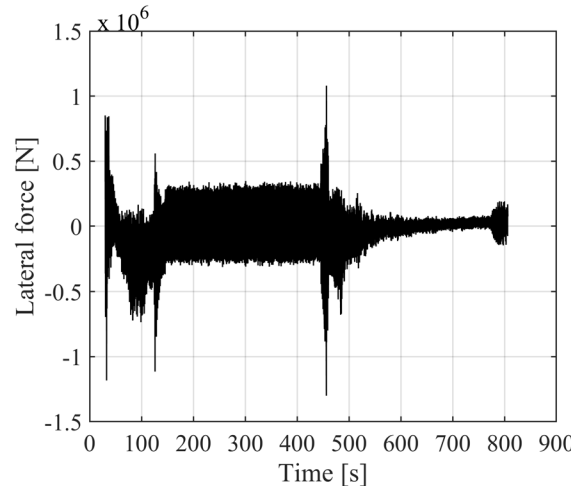
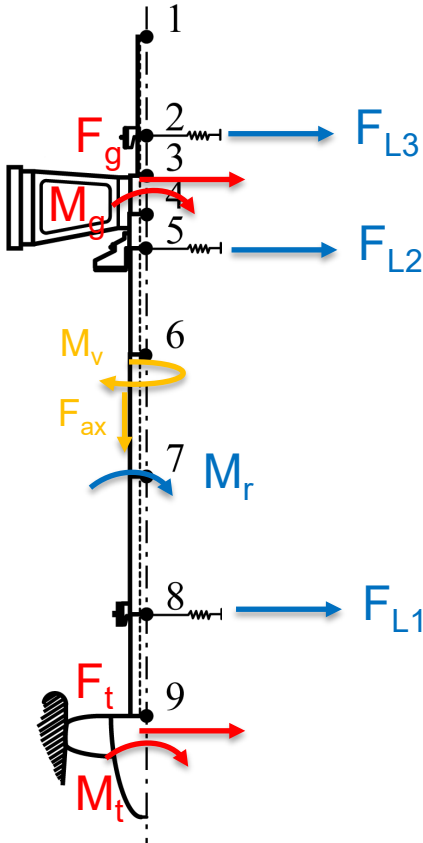
- Rotor Modelling
- Bearing Models for Vertical Rotors
- Electromagnetic Interaction (Uppsala Univ)
- Measuring Techniques (VRD/ Fluid Mech)
- Validations (2 Testrigs at VRD)
- Nonlinear Properties
- Added Turbine Properties (mass/ stiffness/ damping (Fluid Mech)
- Parametric Excitation
- New Components for Added Damping
- Elastic Rotor rim in Generators
- Analytical Bearing Models (speed/excentricity dependent

Measuring Techniques

Quasistatic assumption

- 8 unknown forces
- 8 measuring points

Bearing displacements+ bearing model



Future

- 3D FEM for Generators including Electromagnetic Interaction
- 3D Rotordynamics & Fluid mech % Electro
- Loads on Turbine
- Simulation of Different Operating Conditions
- Life time evaluation



LULEÅ

UNIVERSITY

OF TECHNOLOGY