



**VTT**

# Possible alternatives for shaker table testing in earthquake qualification of I&C cabinets

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VTT – beyond the obvious

# Outline

- Problem statement
- Building plus I&C cabinet interaction
- Traditional earthquake qualification
- Method explored in this study
  - Vertical shaking direction
  - Horizontal shaking direction
  - Re-composing MDOF responses
- Suggested validation path

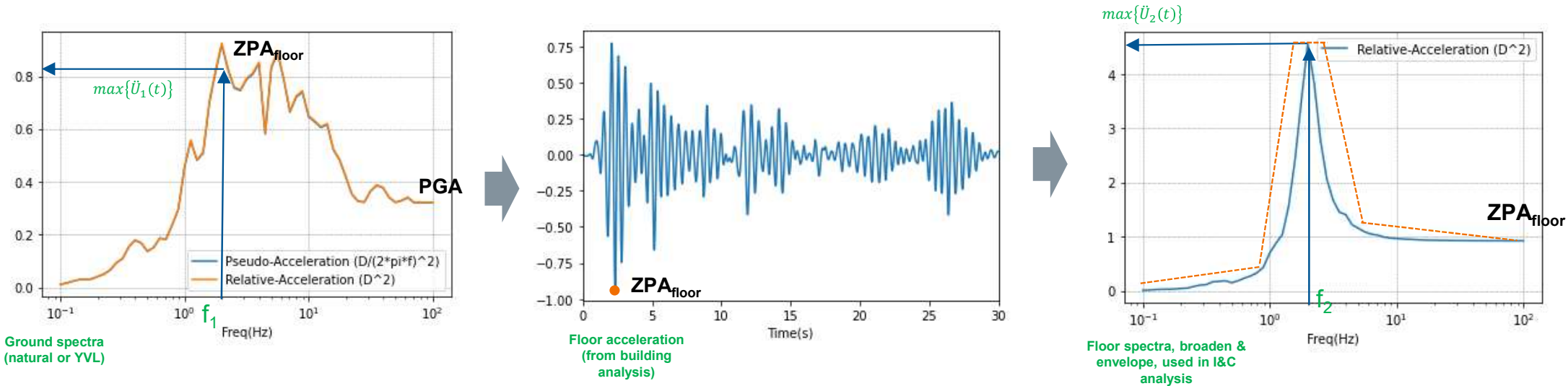
# Problem statement

- **Earthquake qualification** of I&C cabinets is a very costly and has several shortcomings. It involves producing a full-scale mockup and **physically testing it on a shaker-table**.
  - Problem #1: The mockup is not used in the plant; it is especially difficult to reproduce the realistic fixing conditions in the lab (i.e. the boundary conditions in terms of stiffness and interactions).
  - Problem #2: Any change of an operational I&C affects the dynamics; so, ideally a re-qualification would be necessary.
- Research question: Could there be a method, relying more on analysis, which could **replace all or parts of an I&C cabinet earthquake qualification** process?

# NPP building & IC cabinet



- Analysis goes two steps:



- Input to building analysis is a free-field spectra (ground response spectra); a pSA.
- The floor acceleration (ZPA) is obtained from a building model (*in figure only one mode for building is used  $f_1$* ).
- The floor spectra (enveloped and broadened) is input for the I&C analysis (*in figure only one mode for I&C is used  $f_2$* )

## Traditional qualification process

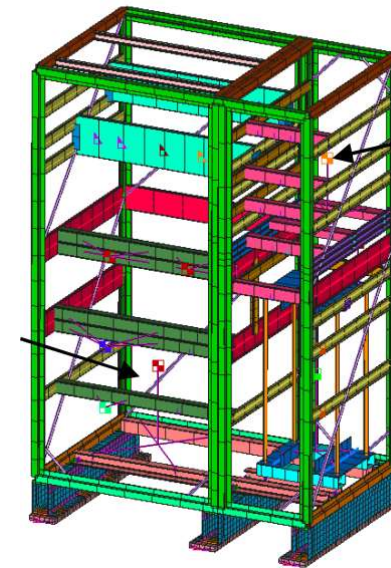
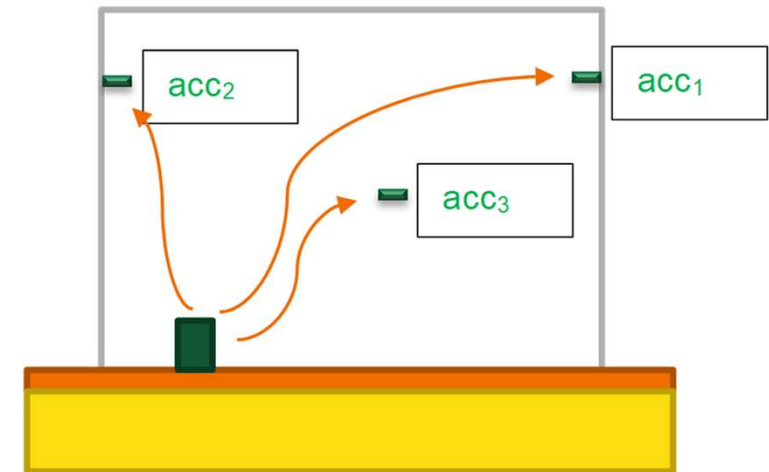
- I&C cabinet **fixed on a shaking table (#1)** in a way equivalent to fixing on the building floor.
- Testing with **sine-sweep at the base**, for mode identification, damping etc.
- Floor response-spectra compatible accelerograms used to qualify the equipment
  - (i) mechanical performance (e.g. degradations etc.) stresses measured
  - (ii) monitor responses of individual equipment (e.g. **displacements (#2)**, accelerations etc.) and
  - (iii) monitoring system functionality.



Ries *et al*, 2017

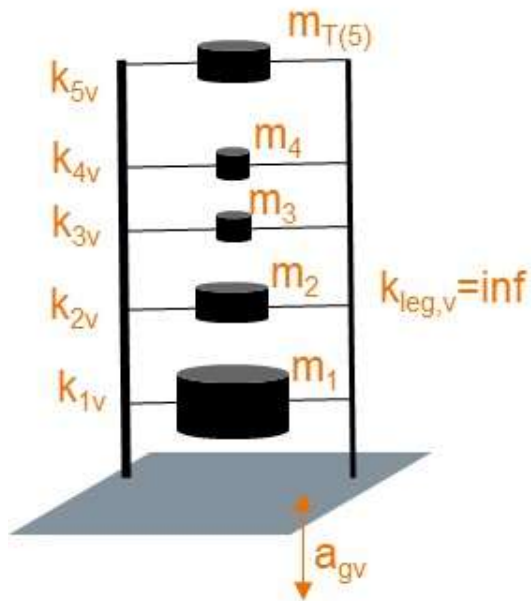
## Research question (expanded)

- Could **qualification in-situ** be carried out, without moving the I&C from its location?
- **Shakers/hammers** could be used for loading; modal properties could be used to calibrate FEM, but we don't want to accept qualification by FEM.
- Re-composing the **dynamic response of MDOFs** to base shaking, without shaking the base...



Ries et al, 2017

# I&C cabinet in vertical direction



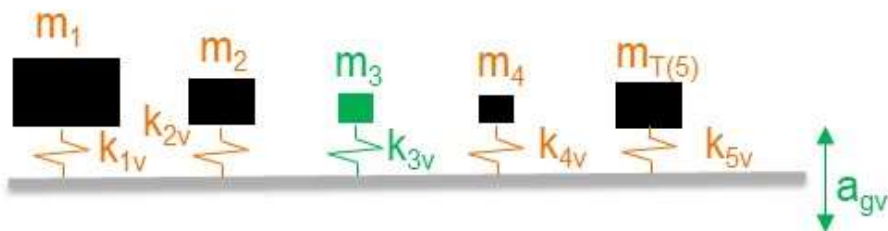
- Vertically for  $m_3$  equations are decoupled:

$$m_3 \cdot U_{g3v}''(t) + k_{3v} \cdot U_{g3v}(t) = -m_3 \cdot a_{gv}(t)$$

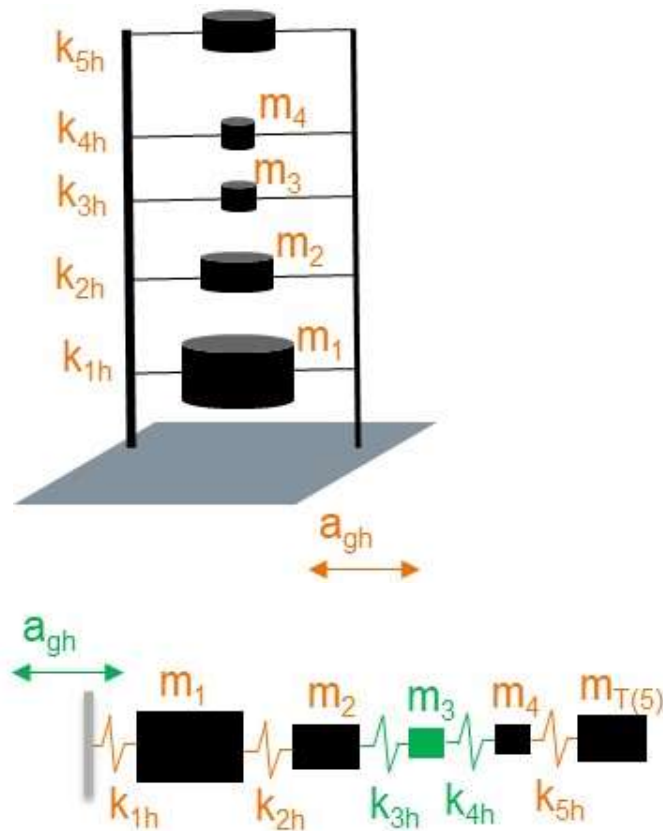
- Green are known:

- $m_3$  from documentation of the I&C, and
- $k_{3v}$  can be calculated from the frequency  $f_{3v}$ , identified from an accelerometer measurement on #3.

- No need to know the rest of the system to calculate e.g.  $U_{g3v}''(t)$ , only the #3 shelve frequency.



# I&C cabinet in horizontal direction



- Horizontally e.g. for  $m_3$  equations are coupled:

$$m_3 \cdot U_{g3h}''(t) - k_{3h} \cdot U_{gh2}(t) + (k_{3h} + k_{4h}) \cdot U_{g3h}(t) - k_{4h} \cdot U_{g4h}(t) = -m_3 \cdot a_{gh}(t)$$

- Greens** can be known; but modal identification of the whole MDOF system needed for  $k_{1h}, k_{2h}, \dots, k_{5h}$ .
- The calculation for  $U_{g3v}$  and  $U_{g3v}''(t)$  involves knowing the motions of other points, i.e. #2 and #4.



# Re-composing of MDOF responses

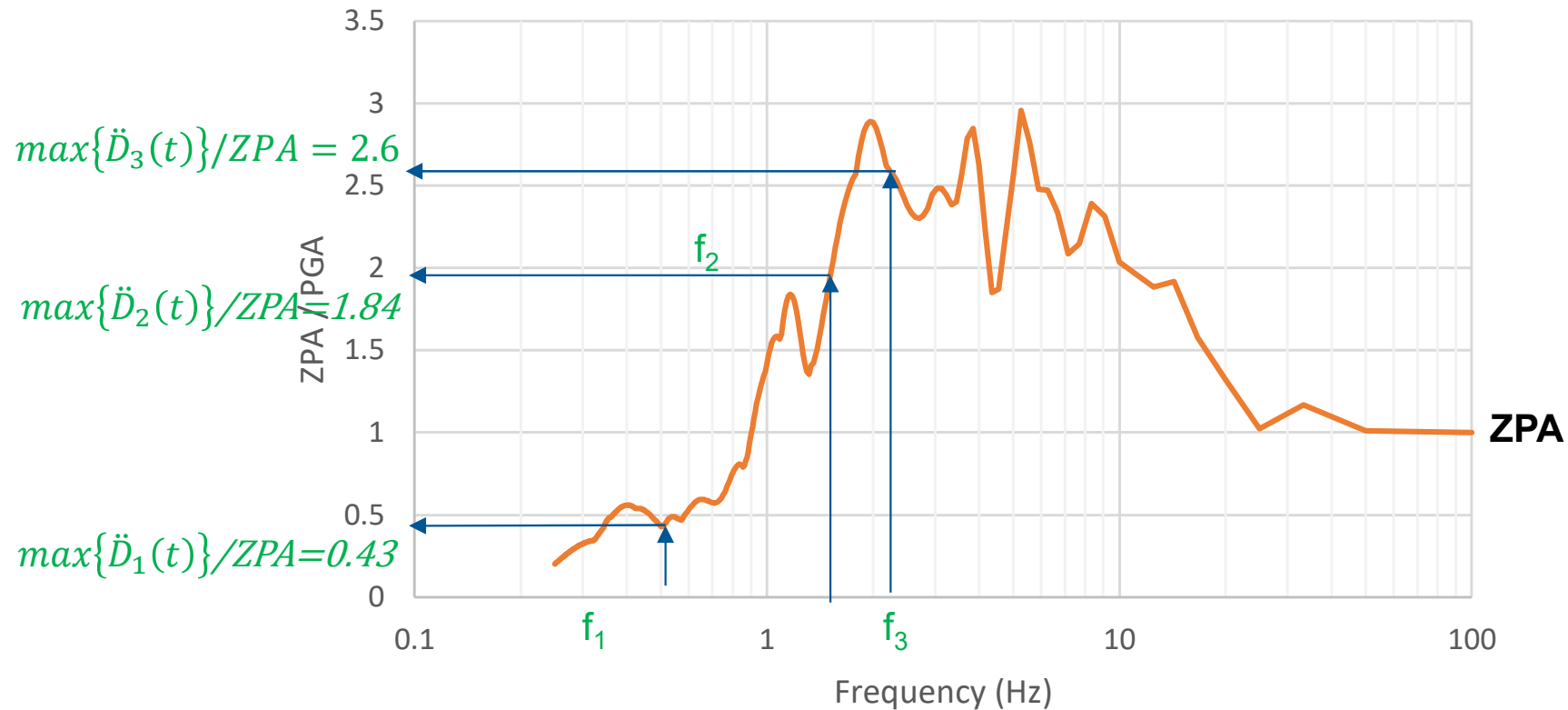
- MDOF response quantity can be calculated from SDOF responses; result is correct if all modal contributions are included. E.g. in cabinet 5<sup>th</sup> DOF model:

$$U_5(t) = \sum_{n=1}^N \Gamma_n \cdot \Phi_{5n} \cdot D_n(t)$$
$$\ddot{U}_5(t) = a_{gh}(t) + \sum_{n=1}^N \Gamma_n \cdot \Phi_{5n} \cdot \ddot{D}_n(t)$$

- Where:
  - $D_n(t)$  and  $\ddot{D}_n(t)$  are the displacement and acceleration of the SDOF's. They are knowns.
  - The MDOF quantities are the participation factor of each mode  $\Gamma_n$  and the mode shape  $\Phi_{5n}$ .

# Focusing on peak quantities simplifies

- Not really interested in  $\ddot{D}_n(t)$ , but in  $\max(\ddot{D}_n(t))$ .
- Can be obtained from the floor-spectra, based on each frequency ( $f_i$ ):



# Summing modal responses approximately

- The max responses are not at the same time so:

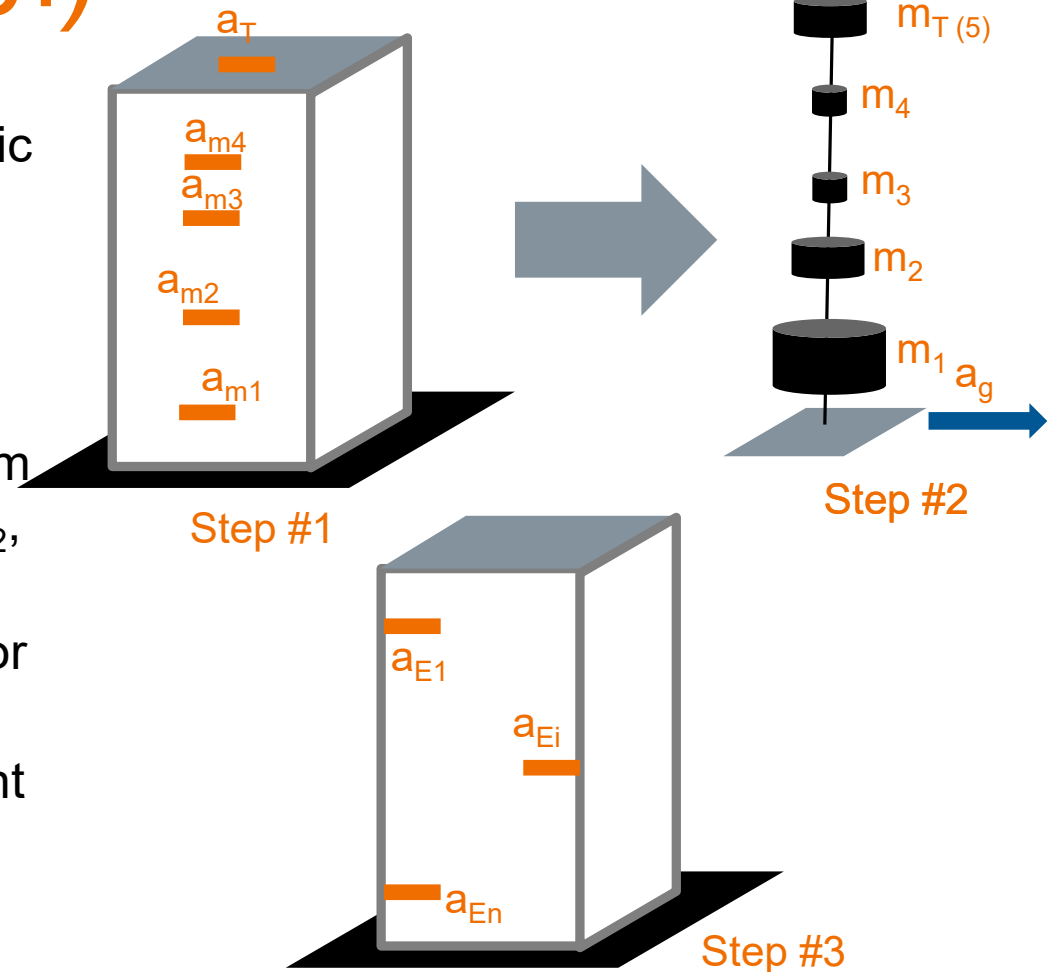
$\max(a_5) \neq \max(a_{5,1}) + \dots + \max(a_{5,5})$ . Approximate solutions:

$$ABS, \max(u_5) \approx \sum_{i=1}^5 \text{abs}(u_{5,i}), SRSS, \max(u_5) \approx \sqrt{\sum_{i=1}^5 (u_{5,i})^2}, CQC$$

- One way is to accept that we need a **simple I&C cabinet model** & work out the dynamics. Not a FEM, but only a “*stick*” model.
- But can we simplify it further for I&C cabinets?!

# Proposed “Conservative response re-composition method (M#5+)”

- No need for shake-table, but a simple dynamic model is needed.
- Step #1: Modal identification with hammer. Obtain modal frequencies ( $f_1, f_2, f_3, \dots$ ), mode shapes and damping.
- Step #2: Estimate masses ( $m_1, m_2, m_3, \dots$ ) from documentations & calculate stiffnesses ( $k_1, k_2, k_3, \dots$ ). Build a simplified dynamic model.
- Step #3: Calculate accelerations from the floor shaking ( $a_g$ ), at each mass location  $m_i$ .
- Step #4: Interpolate acceleration to equipment location  $a_{Ei}$ . Match the  $a_{Ei}$  with component resiliences.



# Simplifying the modal participation factor ( $\Gamma_n$ )

- To calculate it one needs **mode shapes** ( $\Phi$ ), known from the **hammer test** and the distribution of the **masses**  $\bar{r}$  which is problematic; estimates exists at best.

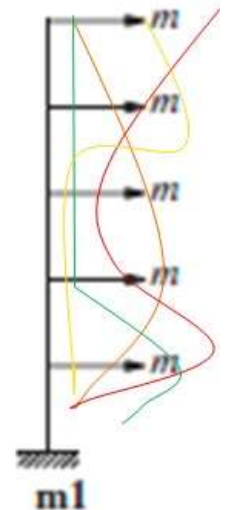
- Calculate the coefficient vector ( $\bar{L}$ )

$$\bar{L} = \Phi^T \cdot M \cdot \bar{r}$$

- and the participation factor ( $\Gamma_i$ ):

$$\Gamma_i = \frac{\bar{L}_i}{\widehat{M_{ii}}}$$

- Monte Carlo simulation could be used to calculate  $\Gamma_i$  with reasonable “m” distributions



## Summary / Suggested validation path

- **Conservative response re-composition (M#5+)** is a promising method to (i) estimate dynamic response of an I&C cabinet and (ii) qualify individual components to acceleration. But it cannot handle (iii) system functionality.
- A simplified **dynamic model is needed**. Could be programmed in PYTHON with hammer test results and masses as inputs. Output is accelerations at equipment locations.
- The **path to validate** the level of conservativeness, would be
  - In step (1) to make comparative studies with sophisticated FEM and
  - In step (2) with shake table tests.

# bey<sup>0</sup>nd

## the obvious

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